AMENDMENTS TO THE SPECIFICATION

Replace the paragraph beginning at Page 1, Line 6 with the following new paragraph:

This application is a continuation of International Application No. PCT/EP02/00354, filed January 16, 2002, <u>now abandoned</u>, which claims priority from German Patent Application No. 10107606.1, filed February 17, 2001. The disclosures of both applications are incorporated herein by reference.

Replace the paragraph beginning at Page 8, Line 23 with the following new paragraph:

The bearing element 2 has a flange portion 13 and a second bearing portion 14. The flange portion 13 forms a second attachment face 15 abutting, in the assembled condition of the flange yoke, the first attachment face 10 of the base element 1 and having a toothing that is formed complementary to the toothing of the first attachment face 10. Furthermore, the flange portion 13 forms a second abutment face 16 extending, corresponding to the first abutment face 12 of the base element 1, parallel to the longitudinal axis 6 and that abuts the same in the assembled condition of the flange yoke. In the second bearing portion 14, a second bearing bore 17 with a second bore axis 18 is provided, wherein the second bore axis 18 is arranged co-axially to the first bore axis 9. The first attachment face 10 and the second attachment face 15 thus provide a means for the transmission of torque around the longitudinal axis.

Replace the paragraph beginning at Page 9, Line 4 with the following new paragraph:

To connect the base element 1 and the bearing element 2 to form a flange yoke, in the flange plate 3 of the base element 1, first through bores 19 are provided that extend parallel to the longitudinal axis 6, and that, starting from the second end face 5, end in the first attachment face 10 and are equally distributed on a partial circumference around the longitudinal axis 6. For each of the first through bores 19, respectively, a first blind hole is arranged in the flange portion 13 of the bearing element 2. The first blind holes extend in extension of the first through bores 19 parallel to the longitudinal axis 6 and start from the second attachment face 15. For

the attachment of the bearing element 2 on the base element 1, first attachment screws 20, such as the illustrated expansion screws, are passed that start from the second end face 5, extend through the first through bores 19, and are screwed into the first blind holes of the bearing element 2. The screw connections with the first through bores 19, with the first blind holes and the first attachment screws 20 are preferably arranged, when viewed in longitudinal direction, in the area of the second bearing portion 14 and, therefore, in the highly loaded area of the flange yoke.

Replace the paragraph beginning at Page 10, Line 17 with the following new paragraph:

To be able to connect the flange element to a mating flange, the second end face 5 forms a connection face 28 with a Hirth-end face serration (partially shown). The base element 1 has through bores 29 for connection screws 30 arranged equally distanced on a partial circumference around the longitudinal axis 6, extending parallel to the longitudinal axis 6 and ending, starting from the connection face 28, in the first attachment face 10. On the residual partial circumference around the longitudinal axis 6, blind holes with internal threads are arranged equally distanced, which also extend parallel to the longitudinal axis 6, start from the connection face 28, and are arranged, when viewed in the longitudinal direction, in the area of the first bearing portion 7. For each of the through bores 29 for the connection screws 30, respectively, a blind hole is arranged in the bearing element 2 that extends in extension of the through bores 29, starting from the second attachment face 15. The blind holes have, respectively, an internal thread. The connection screws 30 can, therefore, either be passed through the through bore 29 of the connection screws 30 and screwed into the blind holes with internal thread of the bearing element 2, or be directly screwed into the blind holes of the base element 1. The connection screws 30 project, thus, beyond the connection face 28 and can be passed through corresponding through bores of a mating flange and can be secured by means of nuts. Preferably, the first attachment screws 20 have screw ends taken up countersunk in the connection face 28. Also, the connection face preferably has a centering means for centering the flange plate relative to a

longitudinal axis of the mating flange. The centering means can be formed by a selfcentering end face serration, especially the illustrated Hirth-end face serration.

Replace the paragraph beginning at Page 11, Line 26 with the following new paragraph:

For the connection of the flange yoke to a mating flange, the base element 201 has through bores 229 for the connection screws 230, which are equally distributed over a circumference around the longitudinal axis 206. A first partial number of the The through bores 229 are arranged, when viewed in longitudinal direction in the area of the first attachment face 210, so as to extend from the connection face 228 to and end in the first attachment face 210. A second partial number of the The through bores 229, which are arranged in the area of the first bearing portion 207 when viewed in longitudinal direction, start from the connection face 228 and end in a first clamping face 234 that is formed by a first recess 235 arranged on a circumference around the longitudinal axis 206. For each of the first partial number of the through bores bore 229 in the area of the first attachment face 210, a through bore 236 is arranged in the bearing element 202 that extends, respectively, in extension of such [[a]] through bores bore 229 of the base element 201. The through bores 236 in the bearing element 202 start from the second attachment face 215 and end in a second clamping face 237 that is formed by a second recess 238 arranged on a circumference around the longitudinal axis 206. The first clamping face 234 and the second clamping face 237 together form an annular clamping face. A first partial number of the The connection screws 230 can, therefore, be either passed through the first partial number of the through bores bore 229 in the base element 201 and through the through bores 236 in the bearing element 202 to the second clamping face 237. A second partial number of the connection screws 230 can be or passed through the second partial number of the through bores 229 bore 234 in the area of the first bearing portion 207 to the first clamping face 234. All of the The connection screws 230 can be screwed into the internal threads 239 of annular elements 240, as shown in Fig. 6. Two annular elements 240 are provided that extend over half of the circumference around the

longitudinal axis 206 and are respectively supported on the first clamping face 234 as well as on the second clamping face 237. The two annular elements 240 abut each other in a plane that is formed by the longitudinal axis and by the bores axes. The annular elements 240 provide the function of absorption of the clamping force in the direction of the longitudinal axis 206, and also the function of radial retainment of the bearing element 202 relative to the base element 201. This is accomplished, on the one hand, by means of a form-fitting contact of the annular elements 240 with the bearing element 202 and the base element 201 and, on the other hand, by means of a frictional contact.

Replace the paragraph beginning at Page 13, Line 5 with the following new paragraph:

Fig. 9 shows the attachment of the bearing element 202 on the base element 201 202 in a cross-sectional view. Components that correspond to components of Figs. Fig. 5 to 8 are provided with the same reference numerals and are as described with Figs. 5 to 8.

Replace the paragraph beginning at Page 13, Line 8 with the following new paragraph:

The bearing element 202 has a first blind hole 241 that is arranged in extension of a first through bore 219 of the base element 201. The first blind hole 241 has an internal thread 242. The screwed-in first attachment screw 220 202 has, in this area, an external thread 243. The first attachment screw 220 abuts, with an end face 244, a base face 245 of the first blind hole 241. The screw connection in the area of the internal thread 242 249 and of the external thread 243 is, therefore, not tensioned. The first attachment screw 220 is only loosely screwed in. The first blind hole 241 continues into a bore portion 246 with an enlarged diameter and without an internal thread. The bore portion 246 ends in the second attachment face 215. In the bore portion 246, a sealing ring 247, in the form of a sealing ring made from an elastomer, is inserted. This also abuts an outer circumferential face 248 of the first second attachment screw 220. The first blind hole 241 is, therefore, sealed to the outside so that the penetration of moisture and corrosion are prevented.

Replace the paragraph beginning at Page 13, Line 21 with the following new paragraph:

Starting from the outer circumferential face 248, the <u>first second</u> attachment screw 220 continues into an expansion portion 249 having a smaller diameter. The expansion portion 249 is arranged within the first through bore 219 of the base element 201. The first through bore 219 merges in a bore portion 250 having an enlarged diameter that ends in the connection face 228. In the bore portion 250, the first second attachment screw 220 has an outer circumferential face in the form of a hexagon 251. Onto the hexagon 251, a sleeve 252 is moved on, wherein the sleeve 252 is supported on a shoulder 253 in the transition area of the first through bore 219 to the bore portion 250 having the enlarged diameter. In continuation to the hexagon 251, the first second attachment screw 220 has a screw end with an external thread 254, onto which a nut 255 is screwed, to be tensioned against the sleeve 252. A ring 256 having an internal hexagon is slid onto the nut 255. The ring 256 is welded to a plate 257. The plate 257 is arranged between the connection face 228 and the ring 256 and has a multitude of bores 258, through which respective first second attachment screws 220 are passed. The ring 256 forms, therefore, together with the plate 257, a retainment against rotational displacement.

Replace the paragraph beginning at Page 14, Line 8 with the following new paragraph:

For detaching of the <u>first second</u> attachment screw 220, the end having the external thread 254 and the screwed-on nut 255 retained by the ring 256, is to be removed by means of a gas cutter. By means of the hexagon 251, the screw 220 can then be removed from the first blind hole 241, wherein because of the sealing of the first blind hole 241, corrosion on the threads 242, 243 is prevented.